
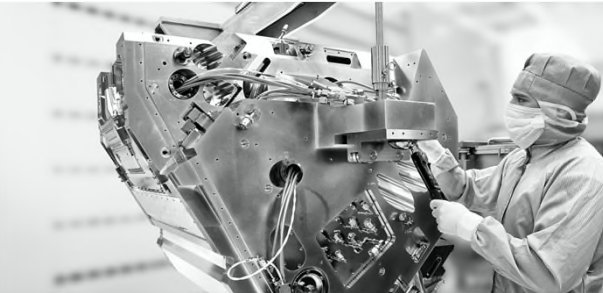


**Thermal Fluid-Structure Analysis**






**Timo Laufer**  
System Engineering  
March, 19th, 2014

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**Thermal Fluid-Structure Analysis of an optical Device including Radiation and Conduction**


Presentation  
March 19<sup>th</sup> – 20<sup>th</sup>, 2014  
euspen  
SIG – Thermal Effects, Zurich, Switzerland

**Timo Laufer**  
Staff System Engineer  
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**Content**




1. Carl Zeiss SMT GmbH and Products
2. Basic Principles of EUVL (Extreme Ultra Violet Lithography)
3. Essential System-Components, thermal Boundary Conditions
4. Heat Transport Mechanisms in rarefied Gases
5. FEM-Model and Results of the Simulations
6. Summary and Benefits

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**The ZEISS Group around the world**  
Fiscal Year 2012/13




 <p><b>Semiconductor Manufacturing Technology</b> €934 million in revenue 2,858 employees</p>	 <p><b>Industrial Metrology</b> €529 million in revenue 2,239 employees</p>	 <p><b>Microscopy</b> €629 million in revenue 2,801 employees</p>
 <p><b>Medical Technology</b> more than 50 €1,025 million in revenue 3,465 employees approx. 24,000 employees</p>	 <p><b>Vision Care</b> €841 million in revenue 5,588 employees</p>	 <p><b>Camera Lenses, Sports Optics, Planatariums</b> €195 million in revenue 767 employees</p>

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**Sites & employees**  
Carl Zeiss SMT GmbH



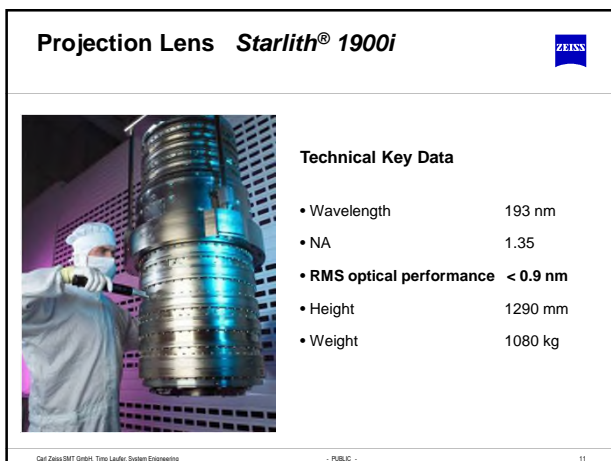
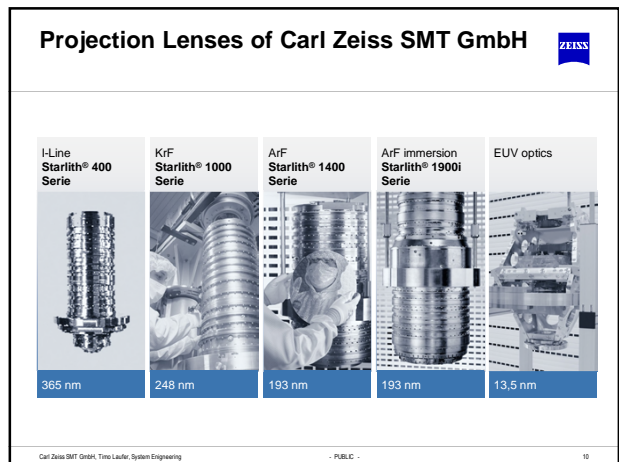
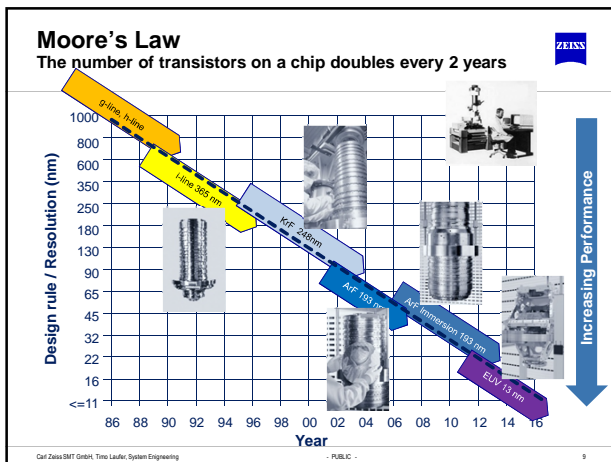
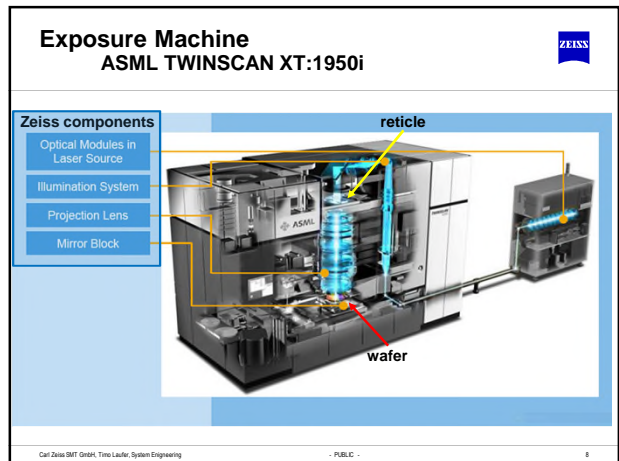
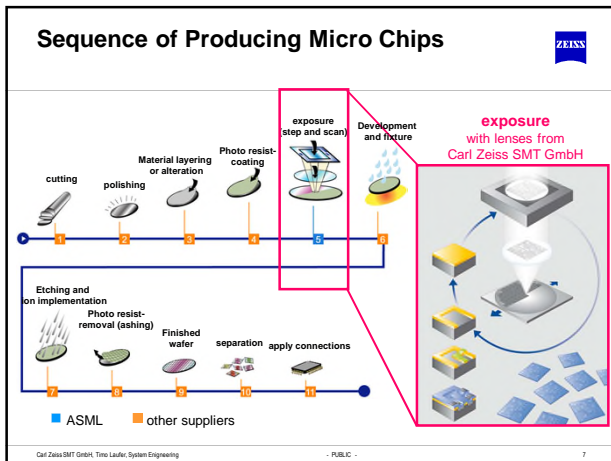
**2,900 employees**

<b>Wetzlar</b> 200 employees	<b>Oberkochen</b> 2,460 employees	<b>Jena</b> 145 employees
<b>Roßdorf</b> 55 employees	<b>Approx. 1,000 in Research &amp; Development</b>	<b>Karmiel, Israel</b> 30 employees

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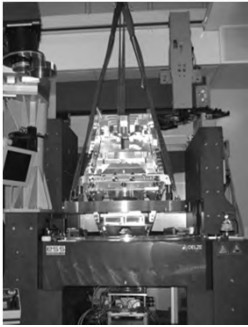


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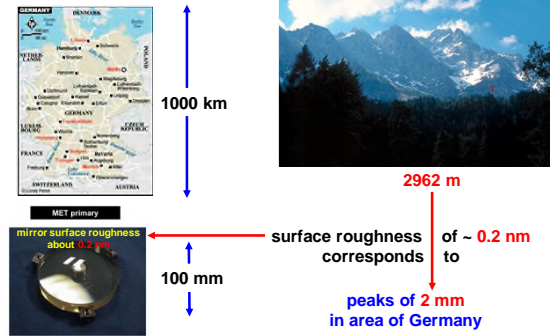
## Projection Optics EUV – Alpha Demo Tool



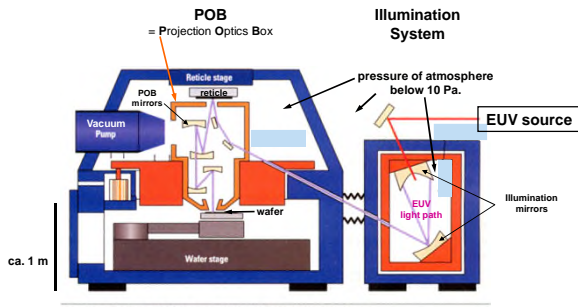
### Key Technical Data

- Wavelength 13.5 nm
- Height 1500 mm
- Weight 790 kg
- Resolution 50 nm (early PO)  
40 nm (AD1, AD2)
- Aberrations < 3 nm (early PO)  
1.4 nm (AD1, AD2)

## EUV Mirror Specs: Compared to Real World



## Alpha-Tool schematic Representation

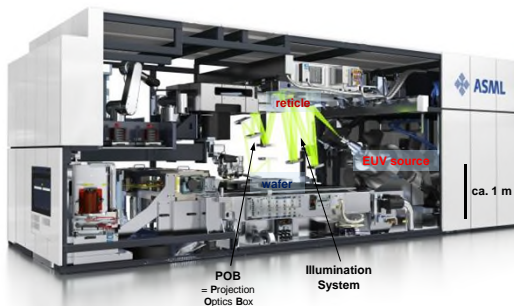


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## EUV-System



## Thermal Boundary Conditions Heat Loads and Heat Sinks



- Heat Loads**
- EUV source compartment, total generated heat flux within the source  $\geq 10$  kW
  - EUV-light absorbed by the mirrors, max. absorbed heat flux  $\geq 500$  W
  - Actuators / motors, sensors, electrical components, ...

- Heat Sinks (Coolers)**
- Water cooled heat sinks to dissipate the heat and
  - heat shields at temperature sensitive components.

## Thermal Boundary Conditions Heat Transfer



### Heat Transfer Mechanisms

- Heat conduction in solids
  - Heat transfer solid / solid (contact resistance)
  - Heat transfer in a rarefied gas (atmosphere in the EUV-system)
    - $p < 10 \text{ Pa}$  →  $0.001 < K_n < 0.1$  → **continuum with slip flow**
      - heat transfer in small gaps
      - fluid / solid interaction
  - IR-radiation with consideration of
    - the view factors of all components
    - the emissivities of all components
- Radiation is important specially for the bigger distances between heat exchanging components and in case of high temperatures; the **hottest** components reach a temperature  **$\gg 700^\circ\text{C}$** .

## Thermal Boundary Conditions Output to be Evaluated



### Output

- **Transient** thermal analyses of the whole EUV system.
  - For the temperature sensitive components a **stability in the  $\mu\text{K}/\text{min}$ -range** must be reached.
- **Transient** thermoelastic analyses show the positioning drift of the optical elements over time.
  - The image position on wafer level **during wafer exposure** must be **smaller than some nanometers**.

## Content



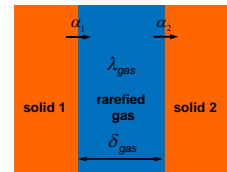
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## Heat Transfer in rarefied gases



$$htc = \frac{1}{R_{th} \cdot A}$$

$$R_{th} = \frac{1}{A_1 \cdot \alpha_1} + \frac{\delta_{gas}}{A_{gas} \cdot \lambda_{gas}} + \frac{1}{A_2 \cdot \alpha_2}$$



- $R_{th}$  : Thermal resistance between heat exchanging solids 1 and 2
- $htc$  : Heat transfer coefficient between heat exchanging solids 1 and 2
- $\alpha_{1,2}$  : Heat transfer coefficient between solid and rarefied gas (atmosphere in the EUV-system)
- $\lambda_{gas}$  : Thermal conductivity of the rarefied gas (atmosphere in the EUV-system)
- $A_1$  : Convection area of fluid / solid boundary 1.
- $A_2$  : Convection area of fluid / solid boundary 2.
- $A_{gas}$  : Surface area of conduction in gas
- $\delta_{gas}$  : Distance between heat exchanging solids 1 and 2

## Sherman-Lees Equation



$$\frac{1}{htc_{gas}} = R_{th} \cdot A = \frac{\delta_{gas}}{\lambda_{gas}} + \frac{\alpha_i + \alpha_j - \alpha_i \cdot \alpha_j}{\alpha_i \cdot \alpha_j} \cdot \sqrt{\frac{\pi \cdot M_{gas} \cdot \bar{T}}{2 \cdot R}} \cdot \left(1 + \frac{\zeta}{4}\right) \cdot p$$

- $htc$  : Heat transfer coefficient between heat exchanging solids 1 and 2
- $R_{th}$  : Thermal resistance between heat exchanging solids 1 and 2
- $\alpha_{i,j}$  : **Accommodation factor between fluid and solids 1 and 2**
- $\lambda_{gas}$  : Thermal conductivity of rarefied gas (atmosphere in the EUV-system)
- $M_{gas}$  : Molecular weight of gas
- $R$  : Universal gas constant
- $\zeta$  : Number of internal energy modes
- $p$  : Gas pressure
- $T$  : Gas temperature
- $\delta_{gas}$  : Distance between heat exchanging solids 1 and 2
- $A$  : Convection area of fluid / solid boundaries 1 and 2

## Continuum with Slip Flow



$$K_n = \frac{l_{mean}}{\delta_{gas}} \quad \begin{array}{l} K_n : \text{Knudsen number} \\ l_{mean} : \text{Mean free path} \\ \delta_{gas} : \text{Distance between heat exchanging solids} \end{array}$$

The Knudsen number  $K_n$  is between  $0.001 < K_n < 0.1$

- **regime**  
continuum with slip flow
- **velocity slip**  
the gas velocity at the wall differs from the wall velocity
- **temperature jump**  
the gas temperature at the wall differs from the wall temperature

The accommodation factors  $\alpha_{i,j}$  in rarefied gases were measured for different materials and different types of machining.

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## FEM Simulations FEM-Model

Summary FEM Data:

- Number of Nodes: > 4 million
- Number of Structure Properties: > 50
- Number of Fluids: 1 (atmosphere)
- Number of Interfaces including thermal resistances: > 300

dummy POB to test the handling

FEM-Model confidential

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## FEM Simulations Results thermal Simulations

dummy POB to test the handling

This FEM-simulation shows **hot spots**, which are short term effects in the beginning of the heating.

The EUV-system has different **time constants**:

- different thermal capacities and
- different thermal resistances from the heat sources to the heat sinks.

The heat loads of some heat generating components are transient, so that we have **transient heat loads as an input for a transient simulation**.

FEM-Model confidential

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## FEM Simulations FE-Model POB Test Facility

schematic representation of the test facility

vacuum chamber

FE-model of the vacuum chamber

POB

photo of the vacuum chamber within the test facility

vacuum chamber

Within the vacuum chamber different scenarios can be tested.

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## FEM Simulations Results POB Test Facility (Vacuum Chamber)

FEM result of the vacuum chamber

Hot spots at the outside of the vacuum chamber can be seen according to the heat generating POB components.

The heat transfer mechanism from the heat generating POB components to the vacuum chamber is mainly radiation and conduction within solids.

with POB inside

photo of the vacuum chamber within the test facility

vacuum chamber

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## Summary and Benefits



### Summary

- Max. absorbed heat loads > 500 W
- Temperatures > 700 °C
- Heat transport mechanisms:
  - heat conduction in solids
  - heat transfer solid / solid
  - heat transfer in a rarefied gas (atmosphere in the EUV-system)  
 $p < 10 \text{ Pa} \rightarrow 0.001 < K_n < 0.1 \rightarrow \text{continuum with slip flow}$
  - radiation, view factors and emissivities of all components are taken into account
- Complex model with > 4 million nodes, > 50 structure properties, > 300 interfaces, ...
- Transient thermal heat loads for transient thermal simulations.
- Thermal stability in the  $\mu\text{K/min}$ - and  $\text{pm/min}$ -range of the EUV-system must be guaranteed  
 $\rightarrow$  **The image position on wafer level during wafer exposure must be stable.**

## Summary and Benefits



### Benefits

- The results of these transient simulations are an important input to quantify the thermal stability of the EUV-system.
- This information is essential during the layout and detailed design phase.
- The results of the transient thermal simulations help in the design of coolers to prohibit hotspots and thermal drifts.
- With a test facility (vacuum chamber) different scenarios of the optics can be tested.

Thank you  
for  
your attention



We make it visible.