



adjustment process, the motor initiates a rotary motion ( $\varphi_1$ ) to couple (s) the tool and the reaction arms to the mechanically sensitive device by a cam mechanism (6) with guiding elements (7). The frictional torque generated by a spring (8) prevents the transmission of a rotary motion to the tool. After rotating the motor shaft of about  $165^\circ$ , a mechanical stop (9,10) at the cam mechanism is engaged. The frictional torque of the spring is overcome and the rotary motion of the tool is enabled ( $\varphi_2$ ). The adjustment mechanism and the device are completely coupled. The angular adjusting motion ( $\varphi_2$ ) is carried out by the driver up to the desired value. After the adjustment process, the motor turns in the opposite direction ( $-\varphi_1$ ) to disconnect (-s) the tool and the reaction arms from the mechanically sensitive device. Another spring (11) is used to keep the cam and the follower in contact.

With this concept, torque compensation is ensured by the simultaneous coupling of the tool and the reaction arms with the mechanically sensitive device. If only one of the two coupling elements would be connected, the motor would be rotated ( $\varphi_3$ ) about its bearing (12) and only the friction torque of the preloaded bearing would be transmitted to the device.

### 3.2 Coupling of adjustment mechanism and device

The couplings of the adjustment mechanism and the mechanically sensitive device, which can be seen in Fig. 2, are designed to avoid overconstraints.

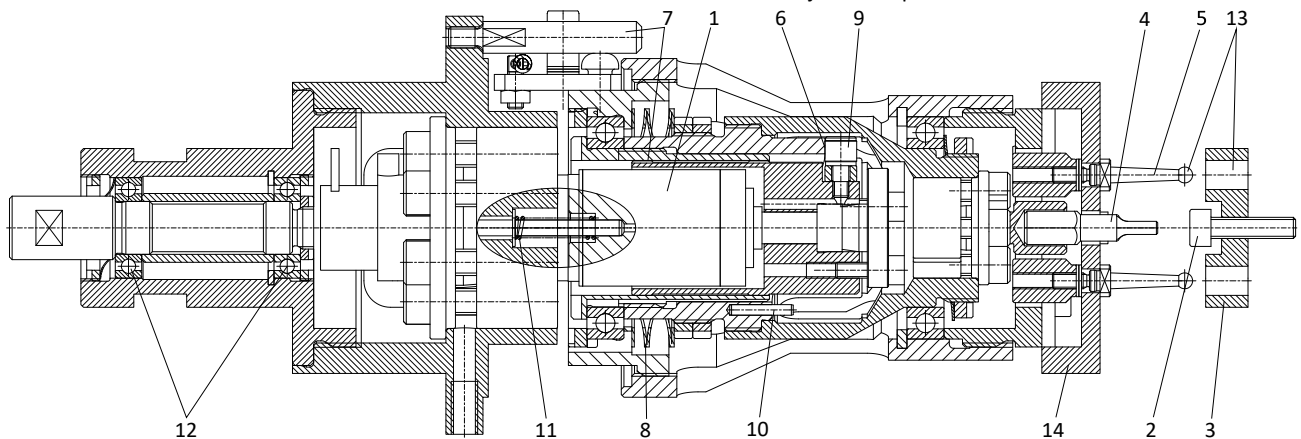


Figure 2. Sectional view of the adjustment drive.

To compensate for lateral and angular misalignment between the adjustment mechanism and the device, sufficient clearance in the form-fit couplings has been provided. To avoid an eccentric torque initiation at the sphere-plane contact pairs (13), an additional elastic element (14), designed as a compliant joint, is attached.

## 4. Results

The adjustment parameters obtained by the prototype are shown in Tab. 1. Restrictions resulting from the realization of the requirements, the operation concept, and the design of the prototype have been considered. The initiated torque cannot be completely compensated by the adjustment drive. Parasitic torques resulting from the preloaded bearing of the motor are applied to the mechanically sensitive device. To create a torque-compensated coupling between the adjustment mechanism and the device, an additional coupling for torque compensation has to be provided. Due to the form-fit pairings of the coupling, there is clearance, which influences the setting accuracy of the adjustment. Owing to the operating principle of the adjustment mechanism a

backlash of  $330^\circ$  exists. The reason for this is the coupling and decoupling of the drive and device. In addition, it is necessary to set a defined position before the start of the adjustment process to enable the form-fitting elements to be coupled. The set position has to be within the range of the tool clearance.

Table 1. Adjustment parameters obtained by the prototype.

Parameter	Value
Parasitic reaction torque	0.25 N mm
Nominal adjustment torque	300 N mm
Resolution adjustment angle	$20'$
Design space	227x80x80 mm
Vacuum conditions	$10^{-6}$ mbar

## 5. Conclusion and outlook

A torque-compensating adjustment mechanism for mechanically sensitive devices is presented as the first step of the ongoing development process. With the technical principle shown in Fig. 1, rotary motions can be transmitted to the device, with low remaining reaction torques of about 0.25 N mm. Overconstraints are avoided due to the design of the coupling. The system can be operated in a vacuum and is completely decoupled from the mechanically sensitive device after the adjustment process.

To verify the obtained results, the designed prototype will be manufactured and tested. In an iterative process, the adjustment mechanism will be optimized in terms of remaining initiated torque, design space, the initial position of the form-fitted couplings and operation in a vacuum. In addition, a drive with higher nominal torque will be introduced to test the prototype as a low-impact fastening system for screws in sensitive precision engineering applications.

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